<u>Appendix 3-2</u> Wilbros Data

COMPRESSOR STATION COST FACTORS

Alaska Natural Gas Development Authority

Prepared for

Michael Baker Jr., Inc.

Anchorage, Alaska

Prepared by

Willbros Engineers, Inc.

Tulsa, Oklahoma



February 8, 2005

"This report is submitted on the basis that Willbros Engineers, Inc. ("Willbros") will only be responsible for liability to the extent directly attributable to errors or omissions of Willbros, but not for any indirect or consequential liability attributable thereto".

Document No.: 51407-001, Rev. 1 Distribution: Project Manager, Project File

TABLE OF CONTENTS

Title Page
Table of Contents
List of Exhibits

Section

Page

1	Intro	oduction	1
2	Com	pressor Station Cost Factors	1
	2.1	Gas Compression Packages	1
	2.2	Refrigeration	2
	2.3	Compressor Stations	3
3	Sum	mary	4



LIST OF EXHIBITS

Exhibit No. Title

1	Compressor Station Cost Factors
2	Solar Taurus Descriptive Literature
2	Solar Titan Descriptive Literature
3	Befrigeration Design Conditions and Peremeters
4	Kenngeration Design Conditions and Parameters



COMPRESSOR STATION COST FACTORS

Alaska Natural Gas Development Authority

1.0 Introduction

The objective of this particular study is to identify "Key Leveraging Design Issues", so that there is a basis for comparing different natural gas pipeline/compressor station system flow study configurations and selecting one configuration for going forward with a feasibility study. The estimates for the different configurations are rough, not for financing purposes, but rather to identify relative costs between the different alternatives. This preliminary cost estimate is +/- 30%.

Willbros prepared an initial estimate of a natural gas pipeline/compressor station system in Alaska in early 2000. Much of this data was referred to throughout the preparation of this report.

2.0 Compressor Station Cost Factors

2.1 Gas Compression Packages

Cost Estimate Assumptions:

- 2.1.1 Compression equipment pricing is based on updated pricing received from Solar Turbines, Inc. for skid mounted packages in the 7,500; 10,000; 15,000 and 20,000 HP ranges. The Taurus package can be used for the 7,500 and 10,000 HP ranges and the Titan package can be used for the 15,000 and 20,000 HP ranges.
- 2.1.2 Since the original proposal Solar has developed a C51 compressor that they have selected to meet the HP requirements. It comes in different staging capability so should fit the needs of each site based on a review by Solar of previous conditions.
- 2.1.3 Pricing for the 15,000 and 20,000 HP ranges is based on two options: the Mars 100 on a Titan skid or the Titan 130 on a Titan skid and includes the Solar C51 compressor. A Mars engine could be replaced with a Titan engine at a later date should the horsepower requirements increase to that level.
- 2.1.4 The budget prices are based on purchase by April 2008 and shipment by April 2009. Should the dates need to be moved out, an escalation factor of 3.5% per year can be applied.



- 2.1.5 Vendor data is included for both the Taurus and the Titan packages as Exhibits 2 and 3.
- 2.1.6 Spares include both commissioning and two year operating and pricing was increased by 20% over original estimate per Solar's recommendation.
- 2.1.7 The same compressor package spares costs are used for all four packages regardless of HP range. Freight costs from San Diego, CA to the vicinity of Fairbanks, AL for both the Taurus and Titan packages have been estimated by Solar.
- 2.1.8 Construction man-hour and cost estimates were provided by Willbros construction estimating personnel.

2.2 Refrigeration

Cost Estimate Assumptions:

- 2.2.1 Randall Gas Technologies provided the original estimate so was consulted in the preparation of this estimate.
- 2.2.2 The refrigeration estimate is based on the 1000 MMscf/d flow case provided by Ward Whitmore. Refrigeration design conditions and parameters are summarized in Exhibit 4.
- 2.2.3 The estimate assumes two refrigeration trains with each train sized for 500 MMscf/d.
- 2.2.4 Refrigeration compression includes one each 100% compressor for each train plus a common spare unit.
- 2.2.5 Refrigeration compression is based upon gas driven engines.
- 2.2.6 Other Equipment/Building costs include the cost of process equipment and a building. Cost for major electrical or instrumentation equipment is included in the installation cost.
- 2.2.7 Installation cost is based upon in-house historical data and includes contractor cost associated with engineering, project management and administration, distributive or bulk materials and equipment (electrical, instrumentation, pipe, valves, fittings, insulation, etc.), procurement activities, construction labor, construction overheads and construction equipment, and contractor overhead and profit.
- 2.2.8 Costs associated with construction camp facilities have not been included.
- 2.2.9 Power is assumed to be available from either an off-site supplier or on-site generation.



- 2.2.10 It is assumed that refrigeration equipment (as appropriate) will be installed in a building due to ambient conditions. The cost of a building has been allowed for in the estimate; however no provisions for any type of building heating system have been included.
- 2.2.11 Refrigeration plant footprint size estimate is 175 ft x 300 ft max.
- 2.2.12 Equipment pricing based on Randall's in-house historical data.

2.3 Compressor Stations

Cost Estimate Assumptions:

- 2.3.1 No personnel housing/accommodations or maintenance warehouses will be required at the stations.
- 2.3.2 Material pricing used on the original estimate was revisited and major engineered equipment was requoted where possible. All other material was priced based on either applying an escalation factor appropriate for that type of material or in-house historical data.
- 2.3.3 The typical station model assumes one launcher, one receiver, two scrubbers, two compressor packages and refrigeration. Pricing for the launcher, receiver, scrubbers, piping, valves, erected buildings, concrete, gravel, etc. is included in the station equipment/materials pricing.
- 2.3.4 The method of electrical power supply varies greatly among various station locations throughout Alaska. Electrical materials costs were estimated for each site and ranged from 1.5 to 7 million dollars depending on whether or not generators, substations, transformers and transmission lines are required. Therefore, the total cost for ten stations spaced along an assumed pipeline route from the North Slope to Valdez was averaged to provide a cost for one site.
- 2.3.5 SCADA/Communications costs were included based on an average cost per site.
- 2.3.6 Construction man-hours for one station have been broken down into the following categories: Civil, Mechanical, Electrical/Instrumentation and Supervisory.
- 2.3.7 Construction costs have been calculated based on a \$/man-hour figure for work in Alaska. An escalation factor was applied to the labor rates used in the original estimate.
- 2.3.8 Construction equipment costs are estimated at 25% of construction costs.
- 2.3.9 Annual O&M costs are estimated at 3.5% of initial installation costs. These costs do not include fuel costs.



3.0 Summary

The cost and man-hour estimates are summarized in the attached Compressor Station Cost Factors spreadsheet (Exhibit 1). As indicated on the spreadsheet, the refrigeration installation costs are high when compared to the station and compression package installation costs. However, per Randall Gas Technologies, they are estimated to be within $\pm/-30\%$.



EXHIBIT 1

COMPRESSOR STATION COST FACTORS FOR ALASKA NATURAL GAS DEVELOPMENT AUTHORITY REVISION 1 8-Feb-05

COMPRESSION:

HP RANGE	COMPRESSION PACKAGE (SOLAR)	\$ Equipment	\$ Freight/Crating	\$ Spares/Training	HP	\$/HP	\$ INSTALLATION	TOTAL COST	\$/INSTALLED HP
7,500	TAURUS 60	\$4,230,000	\$80,000	\$57,600	7,700	\$567	\$774,024	\$5,141,624	\$668
10,000	TAURUS 70	\$4,720,000	\$80,000	\$57,600	10,300	\$472	\$774,024	\$5,631,624	\$547
15,000	TITAN COMPRESSOR W/ MARS ENGINE	\$7,450,000	\$110,000	\$57,600	15,000	\$508	\$1,127,219	\$8,744,819	\$583
20,000	TITAN COMPRESSOR	\$7,950,000	\$110,000	\$57,600	20,000	\$406	\$1,127,219	\$9,244,819	\$462

REFRIGERATION:

RANDALL GAS TECHNOLOGIES PACKAGE	\$ Equipment/Materials	\$ Freight	REFRIG HP	\$/HP	\$ INSTALLATI
REFRIGERATION COMPRESSION ONLY	\$8,600,000				
OTHER EQUIPMENT/BUILDING	\$4,700,000				
TOTAL REFRIGERATION	\$13,300,000	\$1,200,000	17,400	\$833	\$22,400,000
	RANDALL GAS TECHNOLOGIES PACKAGE REFRIGERATION COMPRESSION ONLY OTHER EQUIPMENT/BUILDING TOTAL REFRIGERATION	RANDALL GAS TECHNOLOGIES PACKAGE \$ Equipment/Materials REFRIGERATION COMPRESSION ONLY \$8,600,000 OTHER EQUIPMENT/BUILDING \$4,700,000 TOTAL REFRIGERATION \$13,300,000	RANDALL GAS TECHNOLOGIES PACKAGE \$ Equipment/Materials \$ Freight REFRIGERATION COMPRESSION ONLY \$8,600,000 OTHER EQUIPMENT/BUILDING \$4,700,000 TOTAL REFRIGERATION \$13,300,000	RANDALL GAS TECHNOLOGIES PACKAGE \$ Equipment/Materials \$ Freight REFRIG HP REFRIGERATION COMPRESSION ONLY \$8,600,000 \$8,600,000 \$4,700,000 \$1,200,000 \$1,400 TOTAL REFRIGERATION \$13,300,000 \$1,200,000 \$1,4	RANDALL GAS TECHNOLOGIES PACKAGE \$ Equipment/Materials \$ Freight REFRIG HP \$/HP REFRIGERATION COMPRESSION ONLY \$8,600,000 \$8,600,000 \$4,700,000 \$4,700,000 \$4,700,000 \$1,200,000 \$17,400 \$833

OVERALL STATION COST LESS COMPRESSION PACKAGE AND REFRIGERATION:

\$ Equipment/Materials	\$ Freight	\$ Construction Labor	\$ Construction Equipment	TOTAL COST
\$12,557,802	\$133,425	\$7,890,531	\$1,972,633	\$22,554,391

ESTIMATED MAN-HOURS TO CONSTRUCT TYPICAL COMPRESSOR STATION:

				TOTAL
DISCIPLINE	STATION ONLY	ADD TWO TITAN COMPRESSORS	ADD REFRIGERATION	MAN-HOURS
CIVIL	22,926	6,550	25,000	54,476
MECHANICAL	2,560	731	60,000	63,291
ELEC/INSTRUMENTATION	25,014	7,147	35,000	67,161
SUPERVISORY	12,625	3,607	40,000	56,232
TOTAL	63,125	18,035	160,000	241,160

COMPRESSOR STATION ANNUAL OPERATION AND MAINTENANCE COST IS ESTIMATED AT 3.5% OF INITIAL INSTALLED COST.

ION TOTAL COST **\$/INSTALLED HP**

00 \$36,900,000 **\$2,121**

TAURUS70Gas Turbine Compressor Set



General Specifications

Taurus™ 70 Gas Turbine

- Industrial, Two-Shaft
- Axial Compressor
 - 14-Stage
 - Variable Inlet Guide Vanes
 Compression Ratio: 16:1
 - Compression Rat
 Inlet Airflow:
 - 26.5 kg/sec (58.5 lb/sec) – Vertically Split Case
- Combustion Chamber
 - Annular-Type
 - Conventional or Lean-Premixed,
 - Dry, Low Emission (SoLoNO_x^T)
 - 12 Fuel Injectors
- Torch Ignitor System
- Gas Producer Turbine - 2-Stage, Reaction
 - 2-Stage, Reaction
 Speed: 15,200 rpm
- Power Turbine
 - 2-Stage, Reaction
 - Speed: 12,000 rpm
- Bearings
 - Journal: Tilt Pad
 - Thrust: Tilt Pad
- Coatings
 - Compressor Stators and Drums: Inorganic Aluminum
 - Turbine Blades and Nozzles:
 Precious Metal Diffusion Aluminide
- Vibration Transducer Type
 - Proximity Probes

Key Package Features

- Base Frame with Drip Pans
- Driven Equipment Base Frame
 Compressor
 - Compressor Auxiliary Systems
- 316L Stainless Steel Piping ≤4"
- Compression-Type Tube Fittings
- Digital Gauge Panel
- Electrical System Options
- NEC, Class I, Group D, Div. 1 or 2
 CENELEC, Zone 1
- *Turbotronic*[™] Microprocessor Control System
 - Freestanding Control Console
 - Onskid Control System (Div. 2)
 - Color Video Display
 - Vibration Monitoring
 - Gas Turbine and Package Temperature Monitoring
 Historical Displays
- Control Options
- 120-VDC Control Battery/Charger
- Serial Link Supervisory Interface
- Turbine Performance Map
- Printer/Logger
- Process Controls
- Compressor Performance Map
- Compressor Anti-Surge Control
- Field Programming

- Start Systems
 - Pneumatic
 - Direct Drive AC
- Fuel System
 Natural Gas
- Integrated Lube Oil System
 - Turbine-Driven Accessories
 - Tank Vent Separator
 - Flame Trap
- Oil System Options
 - Oil Cooler
 - Oil Heater
- Compressor Cleaning System Options
 On-Crank/On-Line
 - Cleaning Tank
- Gearbox (if applicable)
 Speed Increaser
- Air Inlet and Exhaust System Options
- Enclosure and Associated Options
- Factory Testing of Turbine with Package and Compressor
- Documentation
 - Drawings
 - Quality Control Data Book
 - Inspection and Test Plan
 - Test Reports
 - Operation and Maintenance Manuals

TAURUS 70Gas Turbine Compressor Set

Performance

No Inlet/Exhaust Losses, Relative Humidity 60%, Natural Gas Fuel with LHV = 31.5 to 43.3 MJ/nm³ (800 to 1100 Btu/scf) Optimum Power Turbine Speed Thermal Efficiency: 34.8%

Nominal Rating – ISO At 15°C (59°F), Sea Level

Output Power 7690 kW (10,310 hp)

Heat Rate 10 340 kJ/kW-hr (7310 Btu/hp-hr)

Exhaust Flow 95 630 kg/hr (210,830 lb/hr)

Exhaust Temperature 495°C (920°F)

Available Power



Package Dimensions



Solar Turbines Incorporated P.O. Box 85376 San Diego, CA 92186-5376 U.S.A.

FOR MORE INFORMATION

Telephone: (+1) 619-544-5352 Telefax: (+1) 619-544-2633 Internet: www.solarturbines.com



Caterpillar is a trademark of Caterpillar Inc. Solar, Taurus, SoLoNO_x, and Turbotronic are trademarks of Solar Turbines Incorporated. Specifications subject to change without notice. Printed in U.S.A. © 2002 Solar Turbines Incorporated. All rights reserved. DS70CS/702/1M

Solar Turbines

A Caterpillar Company

TITAN 130 Gas Turbine Compressor Set

Oil & Gas



General Specifications

- Titan[™] 130 Gas Turbine
- Industrial, Two-Shaft
- Axial Compressor
 - 14-Stage
 - Variable Inlet Guide Vanes and Stators
 - Compression Ratio: 16:1
 - Inlet Airflow:
 - 47.4 kg/sec (105.2 lb/sec)
 - Max. Speed: 11,215 rpm
- Vertically Split Case
- Combustion Chamber
 - Annular-Type, Conventional or Lean-Premixed, Dry, Low Emission (SoLoNO_x[™])
 - 21 Fuel Injectors (Conventional)
 - 14 Fuel Injectors (SoLoNO_x)
 - Torch Ignitor System
- Gas Generator Turbine
 - 2-Stage, Reaction
 - Max. Speed: 11,215 rpm
 - Thrust Bearing, Active: Tilting-Pad
 - Thrust Bearing, Inactive: Fixed Tapered Land
- Power Turbine
 - 2-Stage, Reaction
 - Max. Speed: 8855 rpm
 - Full Tilting-Pad Thrust Bearing
- Journal Bearings
- Tilting-Pad
- Coatings
 - Compressor: Inorganic Aluminum
 - Turbine and Nozzle Blades: Precious
- Metal Diffusion Aluminide
- Vibration Transducer Type
 - Proximity Probes

Key Package Features

- Driver Skid with Drip Pans
- Driven Equipment Skid
 - Compressor
- Compressor Auxiliary Systems
- 316L Stainless Steel Piping ≤4"
- Compression-Type Tube Fittings
- Gauge Panel Options
- Analog Gauges
- Digital Display
- Electrical System Options
 - NEC, Class I, Group D, Div 1
 - NEC, Class I, Group D, Div 2
 - CENELEC, Zone 1
- Turbotronic[™] Microprocessor Control System
 - Onskid Control System (Div 2)
- Freestanding Control Console
- Color Video Display
- Vibration Monitoring
- Control Options
- 24-VDC Control Battery/ Charger System
- 120-VDC Accessory Battery/ Charger System
- Gas Turbine and Package Temperature Monitoring
- Serial Link Supervisory Interface
- Turbine Performance Map
- Compressor Performance Map
- Historical Displays
- Printer/Logger
- Predictive Emissions Monitoring
- Process Controls
- Compressor Anti-Surge Control
- Field Programming
- Start Systems
- Pneumatic
 - Direct-Drive AC

- Fuel Systems
 - Natural Gas
 - Liquid
 - Dual (Gas/Liquid)
 - Wide Range
- Integrated Lube Oil System
 - Turbine-Driven Accessories
 AC Motor-Driven Accessories
- AC Motor-Driven Access
 Oil System Options
 - Oil Cooler
 - Oil Heater
 - Tank Vent Separator
 - Flame Trap
- Package Skid Design
- Accommodates Mars[®] and Titan Turbines
- Drop-In Lube Oil Tank
- Modularized System Design
- Axial Compressor Cleaning Systems
 - On-Crank
 - On-Crank/On-Line
 - Portable Cleaning Tank
- Gearbox (if applicable)
 - Speed Increaser
- Speed Decreaser
- Air Inlet and Exhaust System Options

Test Reports

Manuals

Enclosure and Associated Options

Quality Control Data Book

Operation and Maintenance

- Inspection and Test Plan

- Factory Testing of Turbine
- and Package
 Documentation
 Drawings

Solar Turbines

A Caterpillar Company

TITAN 130 Gas Turbine Compressor Set

Oil & Gas

Performance

Output Power	14 800 kW (19,830 hp)
Heat Rate	9990 kJ/kW-hr (7065 Btu/hp-hr)
Exhaust Flow	180 100 kg/hr (397,060 lb/hr)
Exhaust Temperature	485°C (905°F)

Nominal Rating – ISO At 15°C (59°F), Sea Level

No Inlet/Exhaust Losses,

Relative Humidity 60%,

Natural Gas Fuel with LHV = 31.5 to 43.3 MJ/nm³ (800 to 1100 Btu/scf)

Optimum Power Turbine Speed

AC-Driven Accessories

Engine Efficiency: 35.7%

15°C (59°F) Turbine Rating Match. Other Turbine Rating Match Points Are Available.



Package Dimensions



DS130CS-003M

Solar Turbines Incorporated P.O. Box 85376 San Diego, CA 92186-5376 U.S.A.

Caterpillar is a trademark of Caterpillar Inc. Solar, Mars, Titan, SoLoNO_x, and Turbotronic are trademarks of Solar Turbines Incorporated. Specifications subject to change without notice. Printed in U.S.A. © 2004 Solar Turbines Incorporated. All rights reserved. DS130CS/1299/2M- REV0804/EO

FOR MORE INFORMATION

Telephone: (+1) 619-544-5352 Telefax: (+1) 619-544-2633 Internet: www.solarturbines.com



EXHIBIT 4

DESIGN CONDITIONS AND REFRIGERATION PARAMETERS

ANGDA Compressor Station Cost Factors

1000
2515
80

Outlet

Pressure – psig	2505
Temperature - deg F	28

Refrigeration Parameters

Condensing Temperature - deg F	90
Evaporator Temperature - deg F	23
Chiller Duty - MMbtu/h	94
Condenser Duty - MMbtu/h	111
Compressor HP - Without Spare	8700

CONSTRUCTION MAN-HOURS BREAKDOWN COMPRESSOR STATION COST FACTORS FOR ALASKA NATURAL GAS DEVELOPMENT AUTHORITY 18-Feb-05

	TOTAL MAN- HOURS	Drivers %	Operators %	Welders %	Helpers/ Laborers %	Supervisory %
<u>Man-hours:</u>						
Civil	54,476	10%	25%	0%	65%	
E & I (See note below)	38,291	0%	20%	0%	80%	
Mechanical	92,161	5%	20%	25%	50%	
Suprv25% of Above	56,232					100%
Total	241,160					
Note: Electrical and Inst installation of conduits, technicians	trumentation r ducts and pu	nan-hours and br lling wire only. 1	reakdown by % fo Ferminations are	or category of w made by electri	orkers are for cal/instrument	